

## 10/11 Lecture outline

- Last time: Ideal gas has  $U = U(T)$ , so  $dQ = C_V dT + p dV$ . Using ideal gas law, get  $dQ = (C_V + nR)dT - V dP$ . Conclude  $C_P = C_V + nR$  for ideal gas. Thus  $C_V = nR/(\gamma - 1)$ ,  $C_P = \gamma nR/(\gamma - 1)$ . Can integrate  $C_V = dU/dT$  to find  $U(T)$  in terms of  $\gamma$ . Note: in general  $\gamma = \gamma(T)$ . We will not assume it is  $T$  independent. Its  $T$  dependence depends on gas type. Write  $\gamma - 1 = 2/f$ . If it is monatomic, find  $f = 3$ , independent of  $T$ . If diatomic, find  $f \approx 3$  for  $T < \theta_{rot}$ , and  $f \approx 5$  for  $\theta_{rot} < T < \theta_{vib}$ , and  $f \approx 7$  for  $T > \theta_{vib}$ , so  $C_V$  is nearly constant, but with occasional jumps. Approximating  $C_V$  as constant, get  $U(T) = C_V T = f n R T$ . But, in what follows here, we'll write general formulae.

- $P, V$  diagrams and ideal gas. Picture of  $\beta = (\partial \ln V / \partial T)_P = 1/T$ , and  $\kappa_T = -(\partial \ln V / \partial P)_T = 1/P$ .

- Now picture  $dQ = C_V dT + P dV = C_P dT - V dP$ . Comparing, see that  $dT = (dQ + V dP) / C_P = (dQ - P dV) / C_V$ , and thus  $nR dQ = C_V V dP + C_P P dV$ . For a quasistatic and adiabatic process, have  $dT = -P dV / C_V = V dP / C_P$ , which integrates to  $PV^\gamma = \text{constant}$ .

So get  $(\frac{\partial P}{\partial V})_{adi} = -\gamma P/V = \gamma (\frac{\partial P}{\partial V})_T > (\frac{\partial P}{\partial V})_T$ . So adiabatic curve has steeper slope than isothermal curve in  $P/V$  diagram. See here  $\kappa_T = \gamma \kappa_{adi}$  (and more generally too).

- Examples of  $\Delta W$  for various processes. E.g. for solid with  $\kappa_T \approx \text{constant}$ , get  $\Delta W \approx -\frac{1}{2} \kappa_T V_{av} (P_f^2 - P_i^2)$ , e.g. 10g Copper, from  $P_i = 1 \text{ atm}$  to  $P_f = 10^3 \text{ atm}$ .

Examples of  $\Delta W$  for ideal gas

1. isothermal:  $\Delta U = 0$ .  $\Delta Q = \Delta W = nRT \ln(V_f/V_i) = nRT \ln(P_i/P_f)$ .
2. isochoric:  $\Delta W = 0$ .  $\Delta Q = \Delta U = C_V \Delta T$
3. isobaric:  $\Delta W = P \Delta V = nR \Delta T$ .  $\Delta Q = C_P \Delta T = (C_V + nR) \Delta T$
4. adiabatic:  $\Delta Q = 0$ .  $\Delta W = -\Delta U = -C_V \Delta T$ .

- Engines. Efficiency  $\eta \equiv |W|/|Q_H|$ . E.g. isothermal expansion of ideal gas:  $|W| = |Q| = nRT \ln(P_i/P_f)$  has  $\eta = 1$ , but this is a one-shot process. Final state differs from initial.

- For an engine, want cyclic process, coming back to starting state, i.e. closed loop in  $P/V$  diagram. For complete cycle,  $\Delta U = 0$  (state variable). Total work of process =  $|W|$  = area enclosed by cycle in  $P/V$  diagram. In process, some heat  $|Q_H|$  is taken out of some hot working substance (e.g. boiler), and then some heat is ejected into cold area (e.g. the smoke going out into the atmosphere).  $|W| = |Q_H| - |Q_C|$ , so  $\eta = 1 - |Q_C|/|Q_H| \leq 1$ . Perfect engine would have  $\eta = 1$ , but this is impossible.

- Refrigerator performance:  $\omega = |Q_C|/|W| = 1/(1 - |Q_C|/|Q_H|)$ . Perfect refrigerator would have  $\omega = \infty$ , but this is impossible.

- Preview of 2nd law: (Claiius) *no device can be made that operates in a cycle and whose **SOLE** effect is to transfer heat from cooler to hotter body.* In other words, no perfect refrigerators. Equivalent to Kelvin statement *It is impossible to construct a device that operates in a cycle and produces no other effect than the performance of work and the exchange of heat with a single reservoir.* In other words, no perfect engines.

- Show that two statements are equivalent: with a perfect engine, could make a perfect refrigerator; and given a perfect refrigerator could make a perfect engine.

- Nothing beats a reversible engine! Because otherwise, in combination with the reversed engine (acting as a refrigerator) would violate Claiius' statement. All reversible engines have the same efficiency.  $\eta \leq \eta_{max} = \eta_{rev}$ .